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DISCRETE BARS, CONDUCTIVE HEAT  
TRANSFER, AND ELASTICITY

by

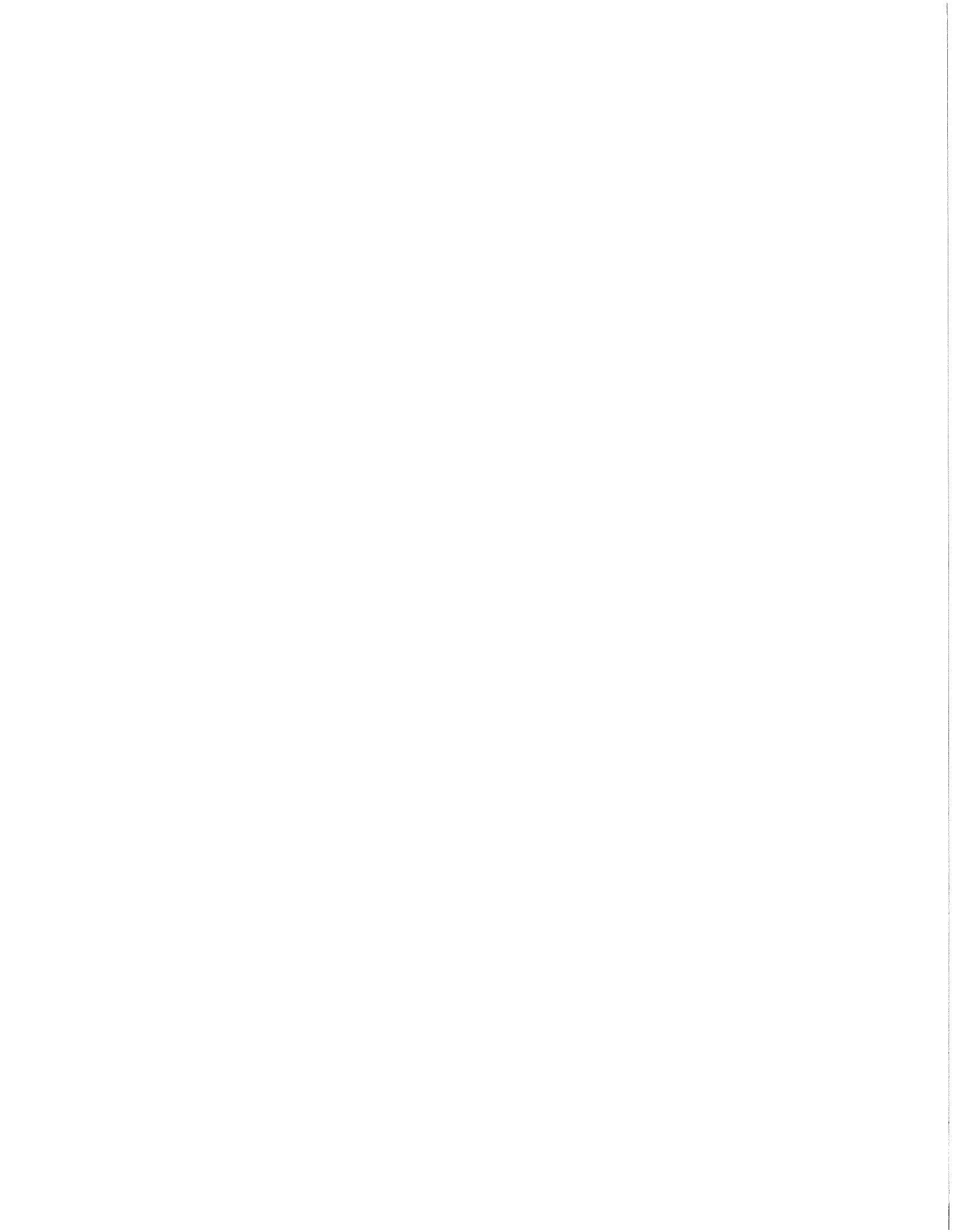
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Appendix: FORTRAN Program for Discrete Conduc-  
tive Heat Transfer, by Sandie Turner Jones

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# DISCRETE BARS, CONDUCTIVE HEAT TRANSFER, AND ELASTICITY

## 1. Introduction

Though the study of bars, or rods, is basic in structural analysis, heat transfer theory, and elasticity theory (see, e.g., refs. [1,3,4, 9-14] and the numerous references contained therein), most of the related models have been continuous and/or linear in nature. The intent of the present paper is to initiate a general computer oriented model which is discrete and nonlinear. For simplicity only, we will restrict attention to two dimensions, and, for convenience, we will describe an arithmetic, energy conserving n-body interaction model first.

## 2. Discrete n-Body Interaction

For positive time step  $\Delta t$ , let  $t_k = k\Delta t$ ,  $k = 0, 1, 2, \dots$ . At time  $t_k$  let particle  $P_i$  of mass  $m_i$  be located at  $\vec{x}_{i,k} = (x_{i,k}, y_{i,k})$ , have velocity  $\vec{v}_{i,k} = (v_{i,k,x}, v_{i,k,y})$ , and have acceleration  $\vec{a}_{i,k} = (a_{i,k,x}, a_{i,k,y})$ , for  $i = 1, 2, \dots, n$ . Position, velocity, and acceleration are assumed to be related by the typical, discrete formulas [5,6]:

$$(2.1) \quad \frac{\vec{v}_{i,k+1} + \vec{v}_{i,k}}{2} = \frac{\vec{x}_{i,k+1} - \vec{x}_{i,k}}{\Delta t}$$

$$(2.2) \quad \vec{a}_{i,k} = \frac{\vec{v}_{i,k+1} - \vec{v}_{i,k}}{\Delta t} .$$

If  $\vec{F}_{i,k} = (F_{i,k,x}, F_{i,k,y})$  is the force acting on  $P_i$  at time  $t_k$ , then force and acceleration are assumed to be related by the discrete dynamical equation

$$(2.3) \quad \vec{F}_{i,k} = m_i \vec{a}_{i,k}.$$

In particular, we will choose  $\vec{F}_{i,k}$  to have a component of attraction which behaves like  $\frac{p}{r^\alpha}$  and a component of repulsion which behaves like  $\frac{q}{r^\beta}$ , where  $p, q, \alpha$  and  $\beta$  are non-negative parameters with  $\alpha \geq 2$ ,  $\beta \geq 2$ , and where  $r$  is the distance between a given pair of particles. For this purpose, let  $r_{ij,k}$  be the distance between  $P_i$  and  $P_j$  at  $t_k$ . Then  $\vec{F}_{i,k}$ , the force exerted on  $P_i$  by the remaining particles, is defined by

$$(2.4) \quad \vec{F}_{i,k} = m_i \sum_{\substack{j=1 \\ j \neq i}}^n \left\{ m_j \left( - \frac{p \sum_{\xi=0}^{\alpha-2} (r_{ij,k}^\xi r_{ij,k+1}^{\alpha-\xi-2})}{r_{ij,k}^{\alpha-1} r_{ij,k+1}^{\alpha-1} (r_{ij,k+1} + r_{ij,k})} + \frac{q \sum_{\xi=0}^{\beta-2} (r_{ij,k}^\xi r_{ij,k+1}^{\beta-\xi-2})}{r_{ij,k}^{\beta-1} r_{ij,k+1}^{\beta-1} (r_{ij,k+1} + r_{ij,k})} \right) (\vec{x}_{i,k+1} + \vec{x}_{i,k} - \vec{x}_{j,k+1} - \vec{x}_{j,k}) \right\}.$$

The particular value of (2.4) lies in the observation that if one defines system work  $W$  from  $t_0$  to  $t_N$  by

$$W = \sum_{i=1}^n \sum_{k=0}^N [(\vec{x}_{i,k+1} - \vec{x}_{i,k}) \cdot \vec{F}_{i,k}],$$

system kinetic energy  $K_k$  at time  $t_k$  by

$$K_k = \sum_{i=0}^n \left[ \frac{1}{2} m_i (v_{i,k,x}^2 + v_{i,k,y}^2) \right],$$

and system potential energy  $V_k$  at time  $t_k$  by

$$V_k = \sum_{\substack{i,j=1 \\ i < j}}^n \left[ \left( -\frac{p}{r_{ij,k}^{\alpha-1}} + \frac{q}{r_{ij,k}^{\beta-1}} \right) m_i m_j \right],$$

then, as in [5] and [6],

$$K_N + V_N = K_0 + V_0, \quad N = 0, 1, 2, \dots,$$

which is the classical law of conservation of energy.

### 3. The Solid State Building Block

In modeling a solid, we will attempt to simulate contemporary physical thought [2,8], in which molecules and atoms exhibit small vibrations within the solid. For this purpose, consider first a system of only two particles,  $P_1$  and  $P_2$ , of equal mass, which interact according to (2.4). Assume that the force between the particles is zero. Then, from (2.4),

$$(3.1) \quad \frac{-p \sum_{\xi=0}^{\alpha-2} (r_{ij,k}^{\xi} r_{ij,k+1}^{\alpha-\xi-2})}{r_{ij,k}^{\alpha-1} r_{ij,k+1}^{\alpha-1} (r_{ij,k+1} + r_{ij,k})} + \frac{q \sum_{\xi=0}^{\beta-2} (r_{ij,k}^{\xi} r_{ij,k+1}^{\beta-\xi-2})}{r_{ij,k}^{\beta-1} r_{ij,k+1}^{\beta-1} (r_{ij,k+1} + r_{ij,k})} = 0.$$

But, if there is zero force between the two particles, then  $r_{ij,k} = r_{ij,k+1}$ , so set  $r_{ij,k} = r_{ij,k+1} = r$  in (3.1) to yield

$$(3.2) \quad \frac{-p \sum_{\xi=0}^{\alpha-2} r^{\alpha-2}}{r^{2\alpha-2}} + \frac{q \sum_{\xi=0}^{\beta-2} r^{\beta-2}}{r^{2\beta-2}} = 0.$$

Thus, for  $\beta \geq \alpha$ ,

$$-pr^{-\alpha}(\alpha-1) + qr^{-\beta}(\beta-1) = 0,$$

or, finally,

$$(3.3) \quad r = \sqrt{\frac{q(\beta-1)}{p(\alpha-1)}}.$$

Consider next a system of only three particles,  $P_1$ ,  $P_2$  and  $P_3$ , of equal masses, whose mutual distances apart are given by (3.3). Since no force acts between any two of the particles, it follows that there is no force acting upon any one of the three. Such a configuration of particles is therefore exceptionally stable and will be called a triangular building block.

When considering a solid we will decompose it into triangular building blocks. In this fashion, the force on any particular particle due to its nearby neighbors will be zero. By an appropriate choice of parameters, the force on any particle due to more distant particles will be made small, thus achieving the small vibrations desired.

To illustrate, let the six particles  $P_1, P_2, P_3, P_4, P_5, P_6$  be located at the vertices of the four triangular building blocks of the triangular region  $OAB$ , shown in Figure 3.1. Assume that  $m_i \equiv 1$ ,  $p = q = 1$ ,  $\alpha = 7$ , and  $\beta = 10$ , so that  $r = \sqrt[3]{1.5}$ . The particles' initial positions are, then,

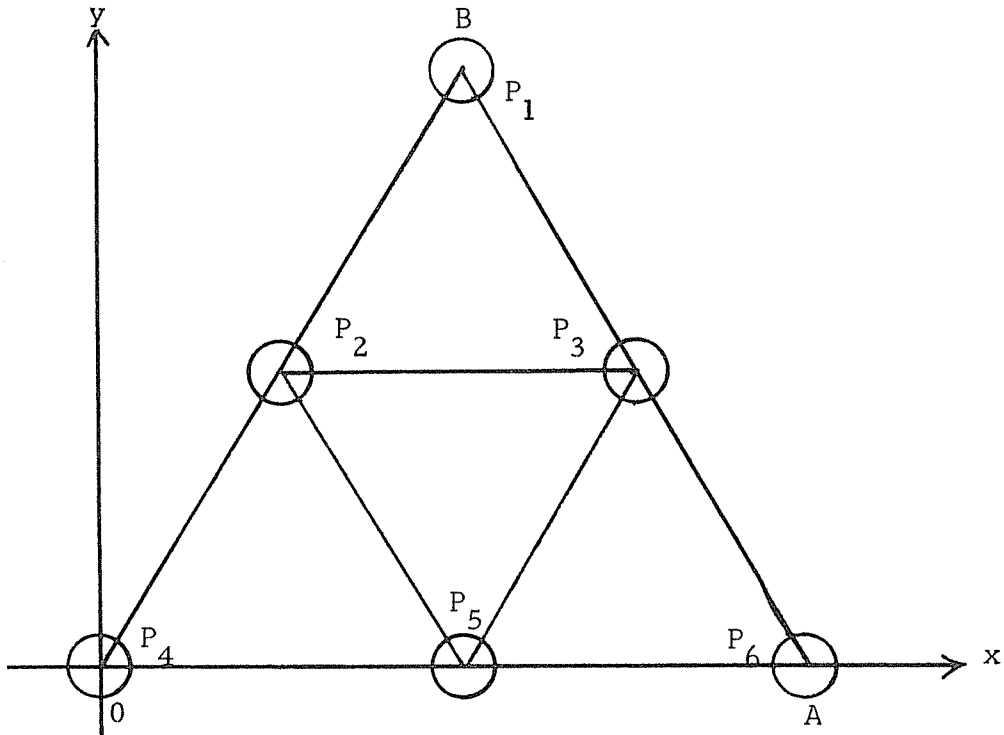


Figure 3.1

$$P_1: (1.14471, 1.98270)$$

$$P_2: (0.57236, 0.99135)$$

$$P_3: (1.71707, 0.99135)$$

$$P_4: (0, 0)$$

$$P_5: (1.14471, 0)$$

$$P_6: (2.28943, 0) .$$

Assign to each particle a  $\vec{0}$  initial velocity. Finally, let particles  $P_4$  and  $P_6$  be fixed and allow the remaining particles to move under force law (2.4). For  $\Delta t = 0.05$  and for 2500 time steps, the motions of  $P_1, P_2, P_3$  and  $P_5$  were generated from (2.1)-(2.4).  $P_1$  and  $P_5$  exhibited small oscillations in the vertical direction only, while  $P_2$  and  $P_3$  exhibited small two dimensional oscillations. The maximum distance, for example, that  $P_1$  moved from its initial position was approximately 0.02, and this occurred at approximately every one hundred time steps. The running time on the UNIVAC 1108 was 4 minutes. The basic computer program used, which is also typical of all examples which follow, was that of Jones [7].

Note that the magnitudes of the oscillations described above can be controlled completely by the appropriate choices of  $p, q, \alpha,$  and  $\beta$ .

#### 4. Flow of Heat in a Bar

Let us now develop the basic concepts of discrete conductive heat



transfer by concentrating on the prototype problem of heat flow in a bar. Physically, the problem is formulated as follows. Let the region bounded by rectangle  $OABC$ , as shown in Figure 3.1, represent a bar. Let  $|OA| = a$ ,  $|OC| = c$ . A section of the boundary of the bar is heated. The problem is to describe the flow of heat through the bar.

Our discrete approach to the problem proceeds as follows. First, subdivide the given region into triangular building blocks, one such possible subdivision of which is shown in Figure 3.2 for the parameter choices  $m_i \equiv 1$ ,  $p = q = 1$ ,  $\alpha = 7$ ,  $\beta = 10$ ,  $a \sim 11$ ,  $c \sim 2$ . Note that from (3.3),  $r \sim 1.1447142426$ .

Now, by heating a section of the boundary of the bar, we will mean increasing the velocity, and hence the potential energy, of some of the particles whose centers are on  $OABC$ . By the temperature  $T_{i,k}$  of particle  $P_i$  at time  $t_k$ , we will mean the following. Let  $M$  be a fixed positive integer and let  $K_{i,k}$  be the kinetic energy of  $P_i$  at  $t_k$ . Then  $T_{i,k}$  is defined by

$$T_{i,k} = \frac{1}{M} \sum_{j=k-M+1}^k K_{i,j},$$

which is, of course, the arithmetic mean of  $P_i$ 's kinetic energies at  $M$  consecutive time steps. By the flow of heat through the bar we will mean the transfer to other particles of the bar of the kinetic energy added at the boundary. Finally, to follow the flow of heat through the

bar one need only follow the motion of each particle and, at each time step, record its temperature.

To illustrate, consider the bar shown in Figure 3.2 with the parameter choices given above, that is,  $m_i \equiv 1$ ,  $p = q = 1$ ,  $\alpha = 7$ ,  $\beta = 10$ ,  $a \sim 11$ ,  $c \sim 2$ . Assume that a strong heat source is placed above  $P_6$ , and then removed, in such a fashion that  $\vec{v}_{5,0} = \left( -\frac{\sqrt{2}}{2}, -\frac{\sqrt{2}}{2} \right)$ ,  $\vec{v}_{6,0} = (0, -1)$ ,  $\vec{v}_{7,0} = \left( \frac{\sqrt{2}}{2}, -\frac{\sqrt{2}}{2} \right)$ , while all other initial velocities are  $\vec{0}$ . With regard to temperature calculation, assume that the velocities of all particles prior to  $t_0$  were  $\vec{0}$ . As regards the choice of  $M$ , which is a difficult choice to make, one would usually wish to choose it relatively large, since the use of an average is, generally, more meaningful when the number of quantities being averaged is relatively large. We shall arbitrarily set  $M = 20$ . From the resulting calculations with  $\Delta t = 0.025$ , Figures 3.3 - 3.7 show the constant temperature contours  $T = 0.1, 0.06, 0.025, 0.002$  at  $t_5, t_{10}, t_{15}, t_{20}$  and  $t_{25}$ , respectively. The resulting wave motion is clear and Figure 3.7 exhibits wave reflection. It is interesting, also, to note that the temperature at  $P_6$  increases, until  $t_{20}$ , at which time it is a maximum, and only then does it proceed to decrease. Figures 3.8 - 3.12 show the constant kinetic energy contours  $K = 0.1, 0.05, 0.01, 0.001$  at each of the times  $t_5, t_{10}, t_{15}, t_{20}, t_{25}$ , respectively, and indicate the magnitude of the particle velocities at these time steps.

Other heat transfer concepts can be defined now in the same spirit as above, as follows. A side of the bar is insulated means that the bar particles cannot transfer energy across this side of the bar to particles outside the bar, while melting is the result of adding a sufficient quantity of heat so that various particle velocities attain sufficient magnitude so as to break the bonding effect of (2.4).

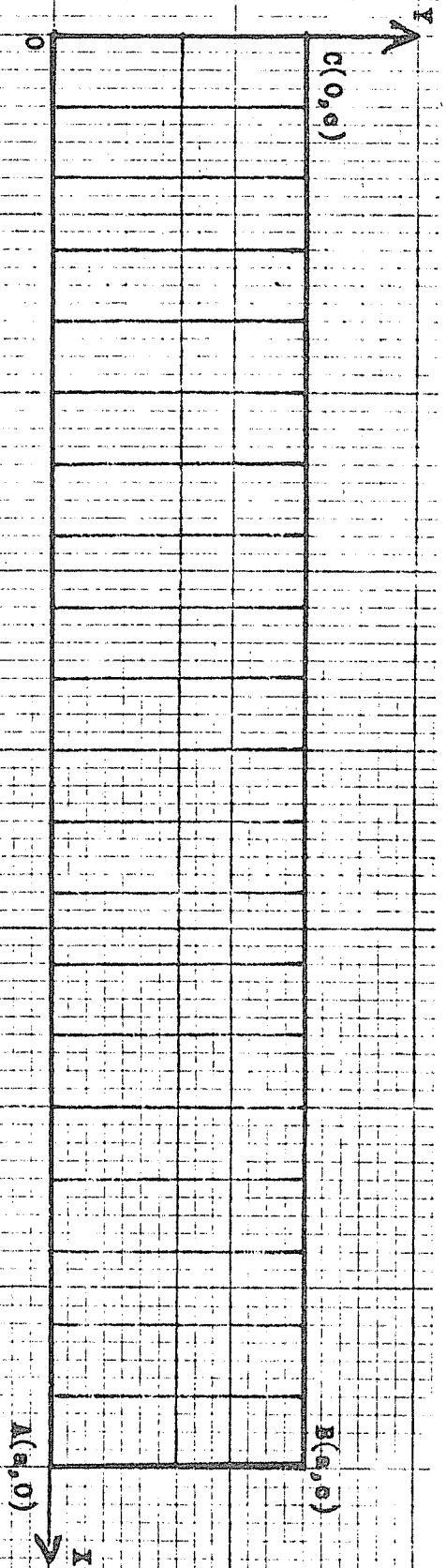


FIGURE 3.1.

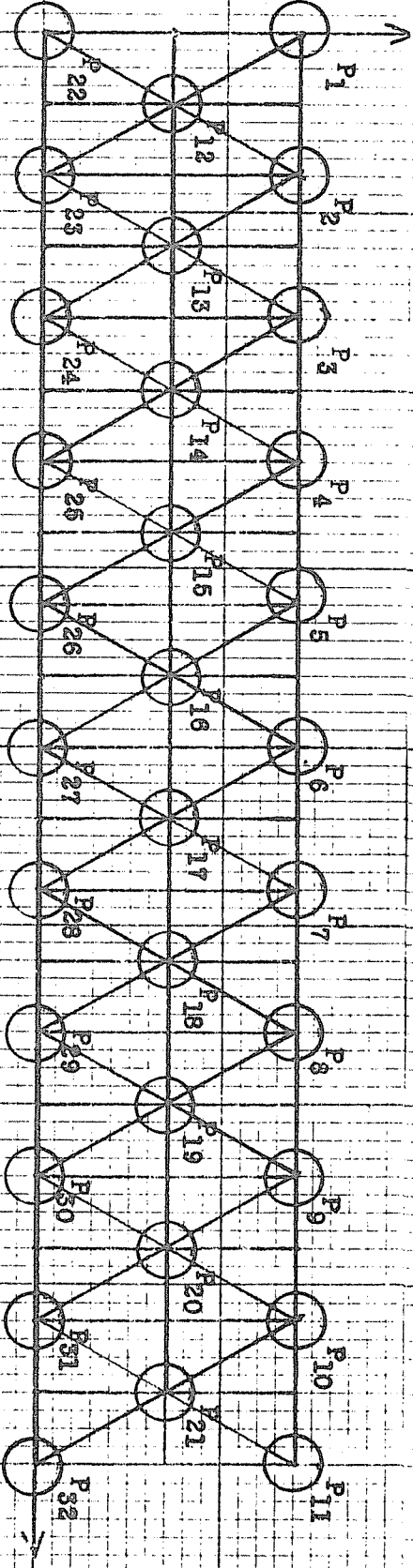


FIGURE 3.2

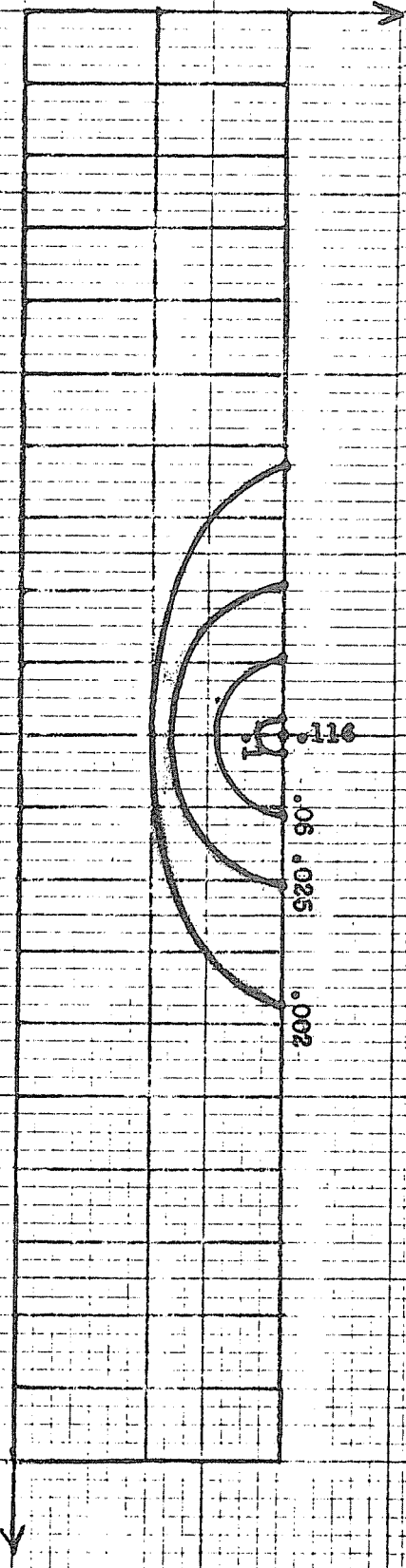


FIGURE 3.3

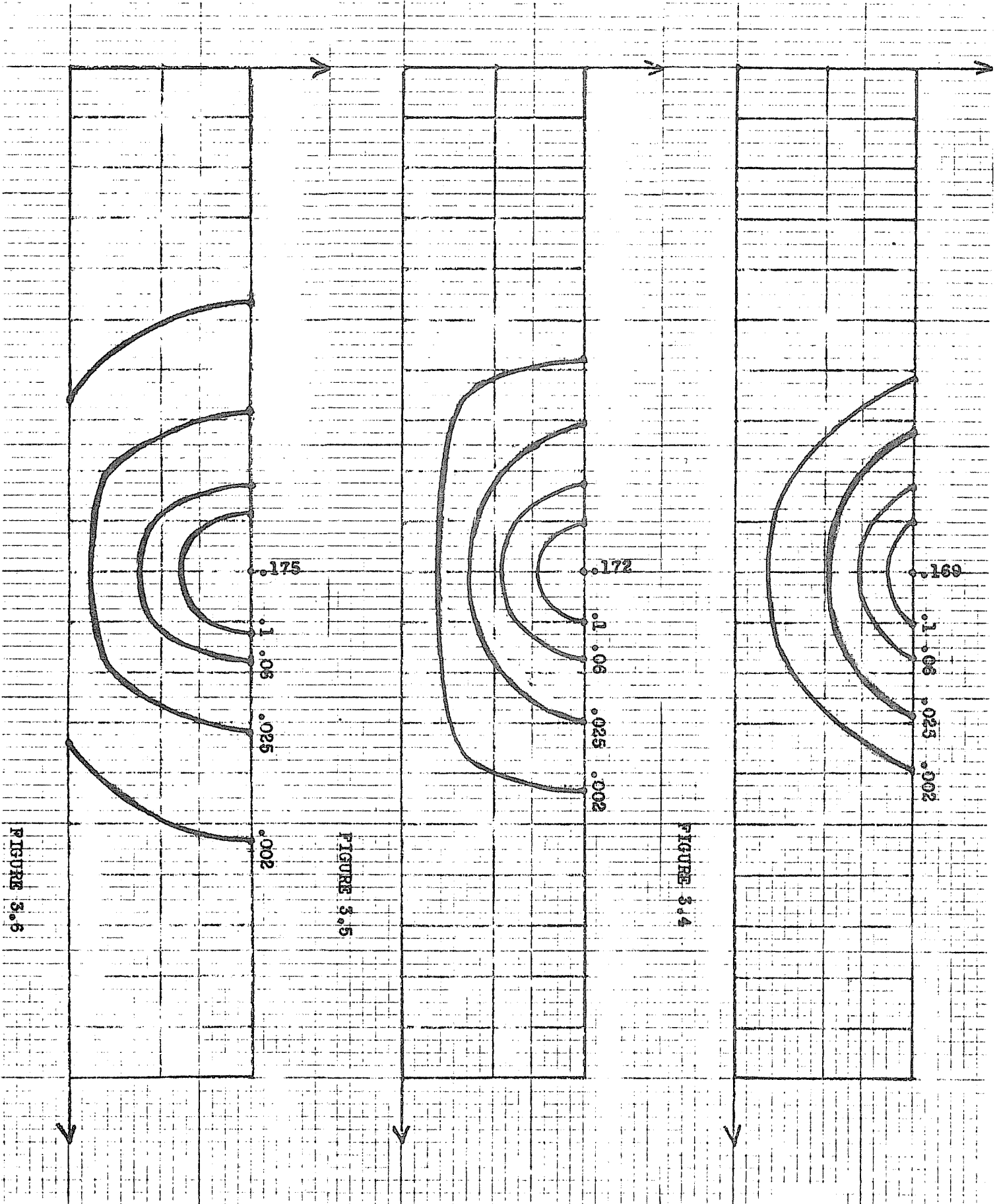


FIGURE 3.4

FIGURE 3.5

FIGURE 3.5

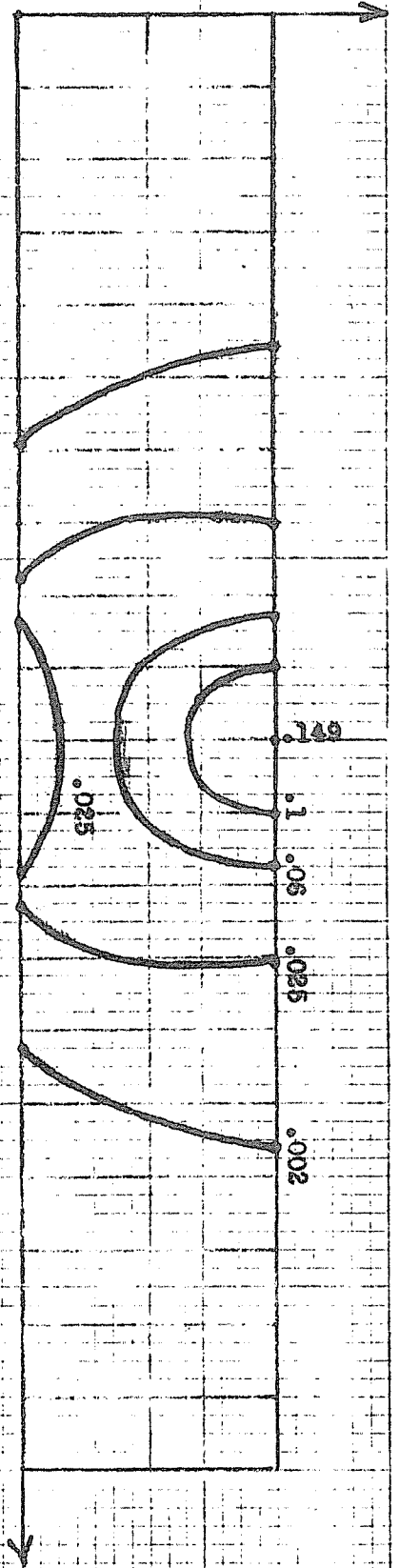


FIGURE 3.7

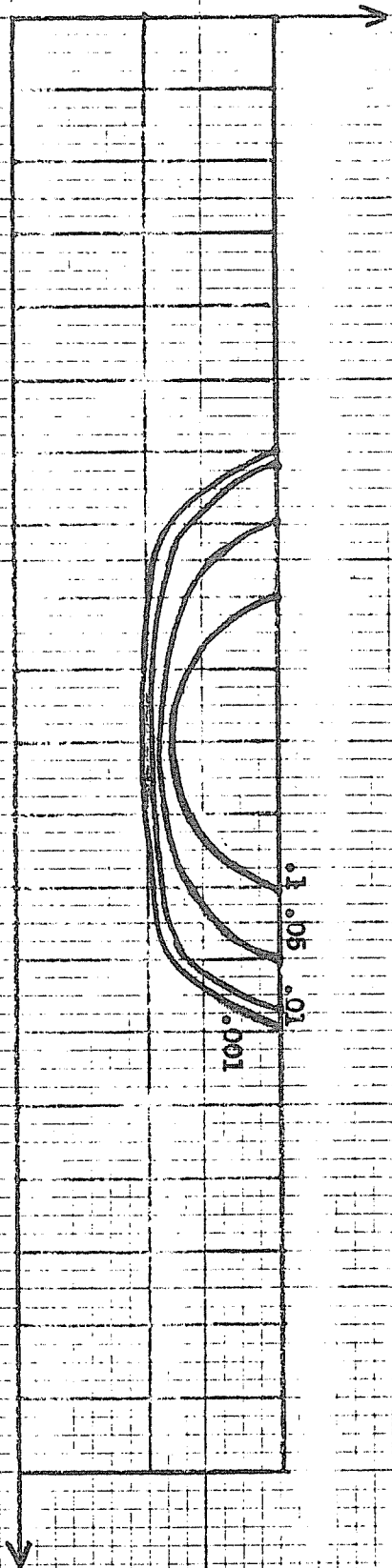


FIGURE 3.8

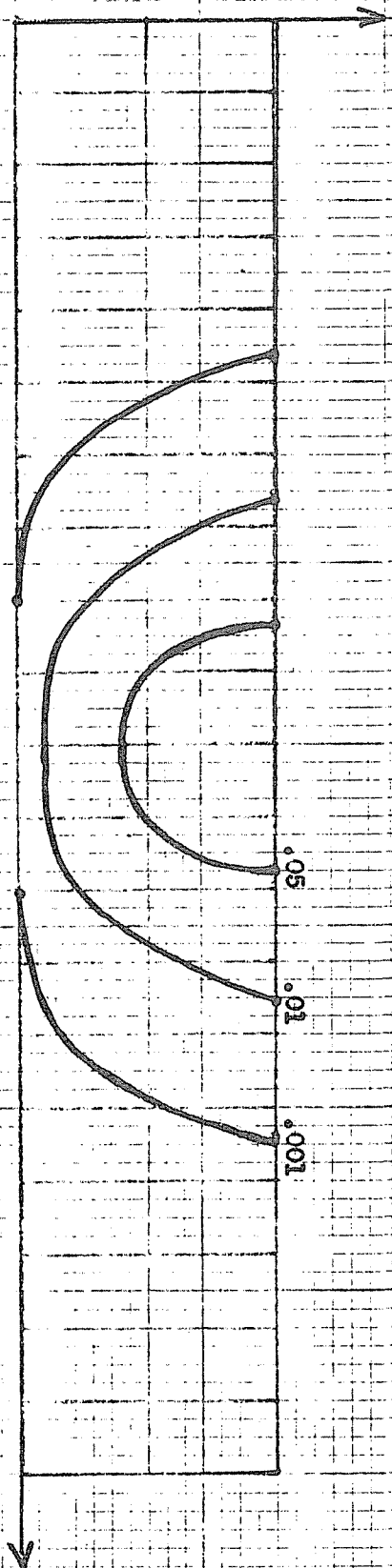


FIGURE 3.9

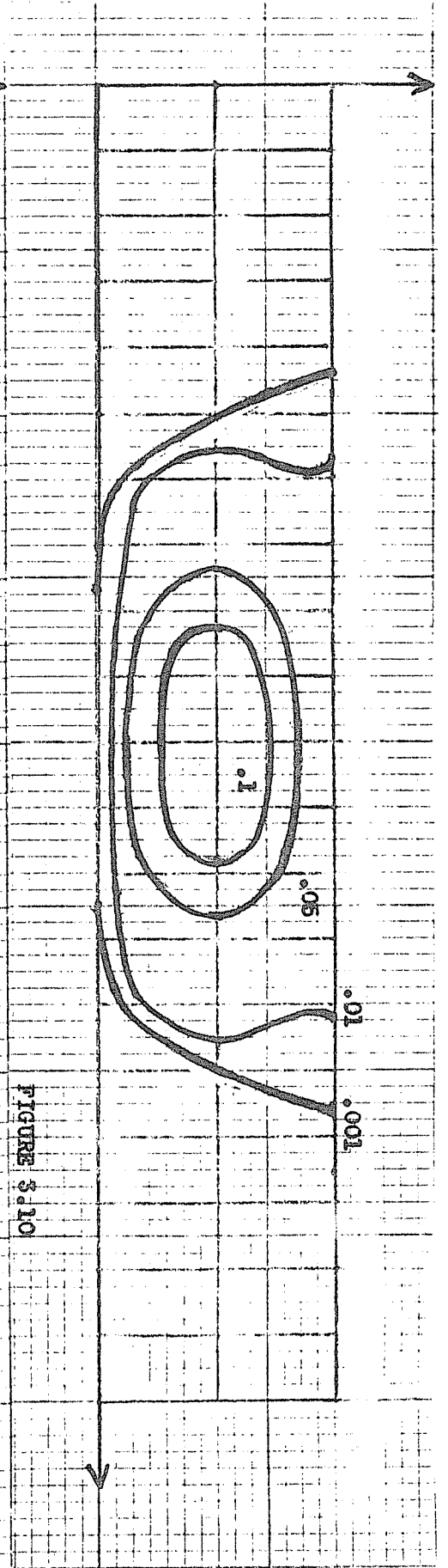


FIGURE 3.10

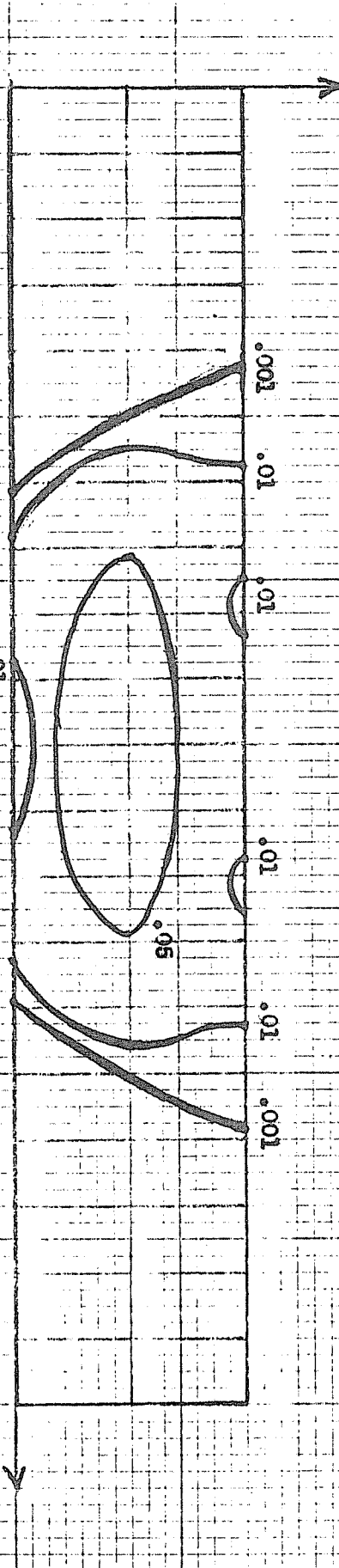


FIGURE 3.11

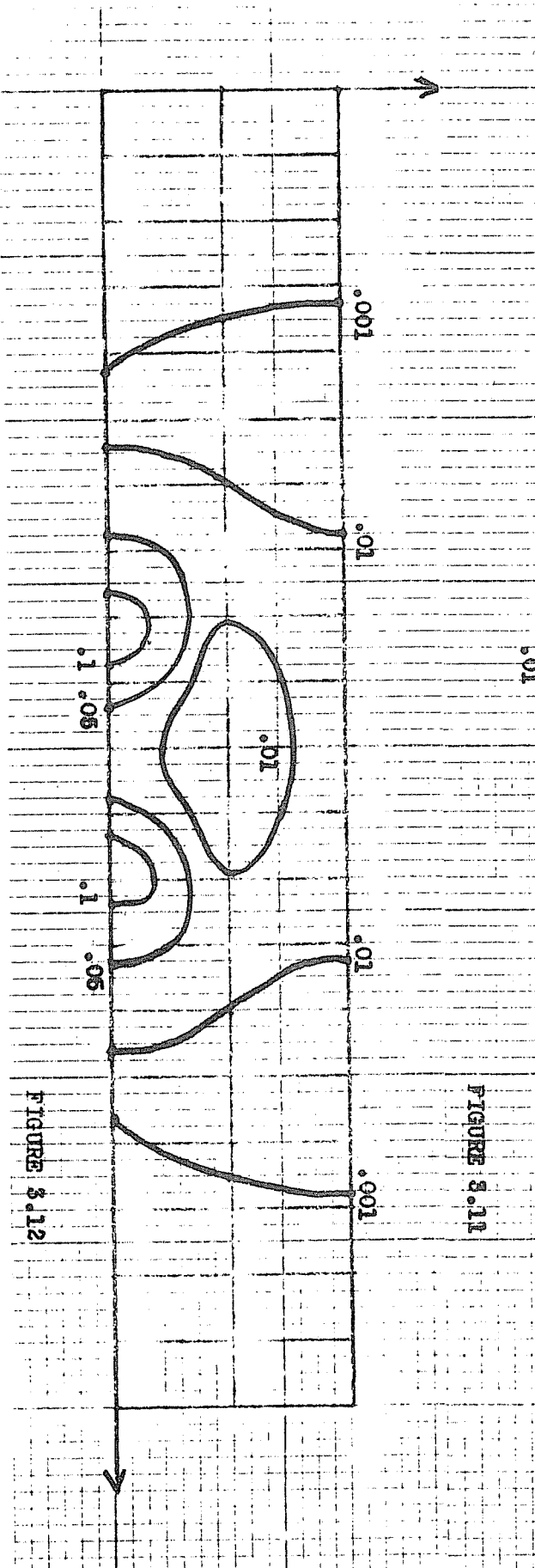


FIGURE 3.12

### 5. Oscillation of an Elastic Bar

Next, let us develop the basic concepts of discrete elasticity by concentrating on the vibration of an elastic bar. The problem is formulated physically as follows. Let the region bounded by rectangle  $OABC$ , as shown in Figure 3.1, represent a bar which can be deformed, and which, after deformation, tends to return to its original shape. The problem is to describe the motion of such a bar after the external force, which has deformed the bar, is removed. Equivalently, the problem is to describe the motion of an elastic bar after release from a position of tension.

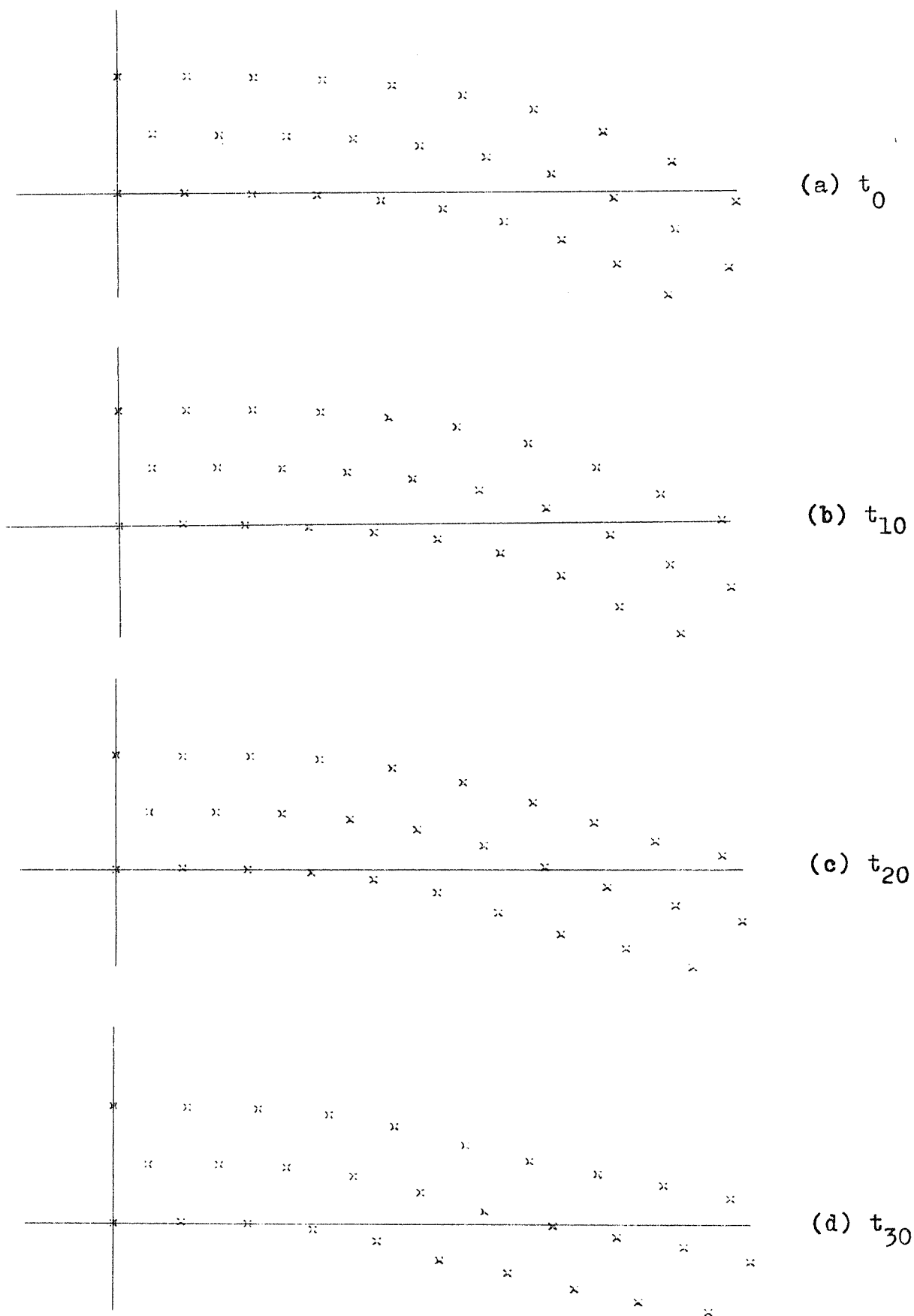
Our discrete approach proceeds as follows. The given region is first subdivided into triangular building blocks. Then, deformation results in the compression of certain particles and the stretching apart of others. Release from a position of deformation, or tension, results, by (2.4), in repulsion between each pair of particles which have been compressed and attraction between each pair which have been stretched, the net effect being the motion of the bar.

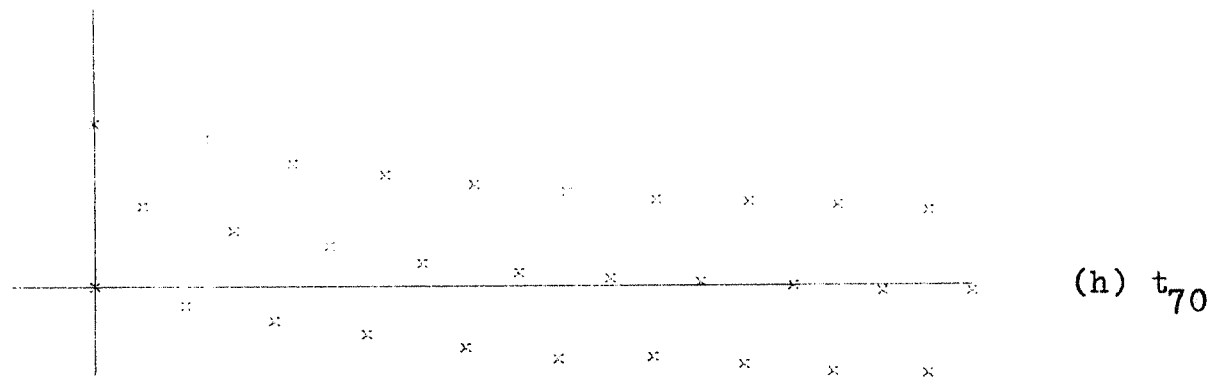
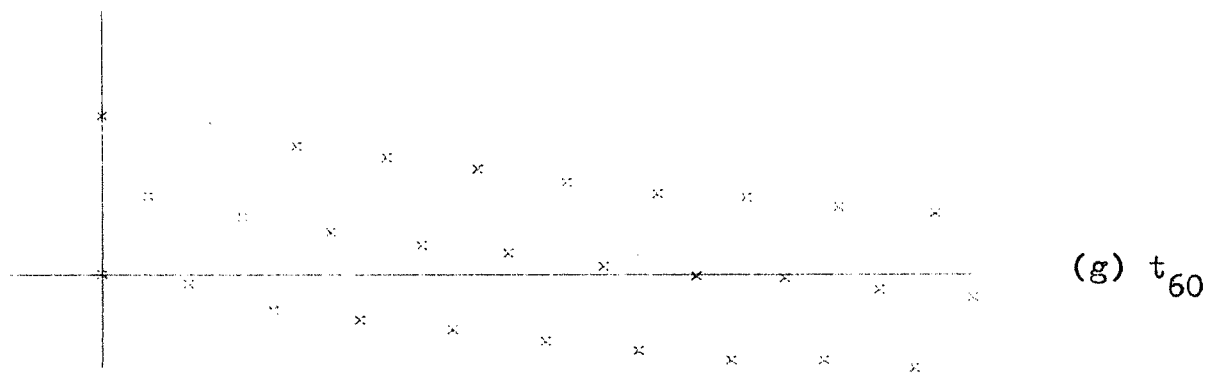
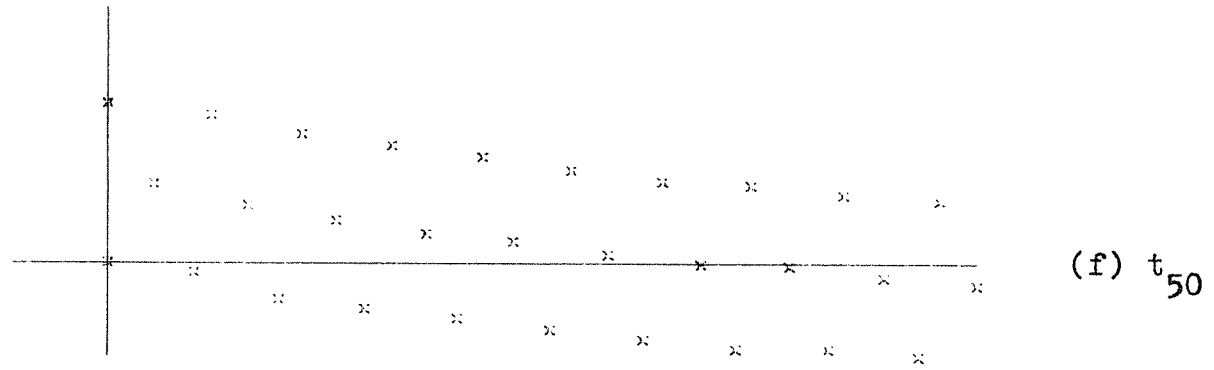
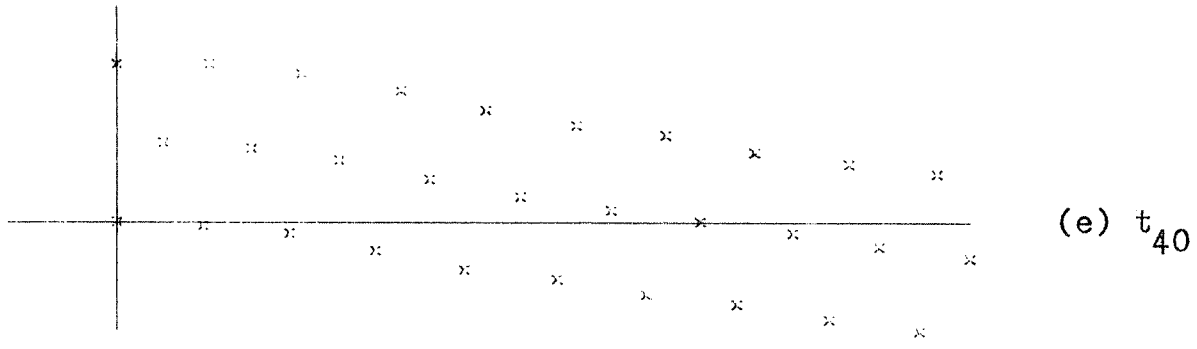
As a particular example, let  $m_i \equiv 1$ ,  $\alpha = 7$ ,  $\beta = 10$ ,  $p = 425$ ,  $q = 1000$ , and  $\Delta t = .025$ . From (3.3),  $r = 1.52254$ . Consider, for variety, the thirty particle bar which results by deleting  $P_{11}$  and  $P_{32}$  from the configuration of Figure 3.2. The particles  $P_1$ ,  $P_{12}$ , and  $P_{22}$ , whose respective coordinates are  $(0, 2.63711)$ ,  $(.76127, 1.31855)$ , and  $(0, 0)$ ,

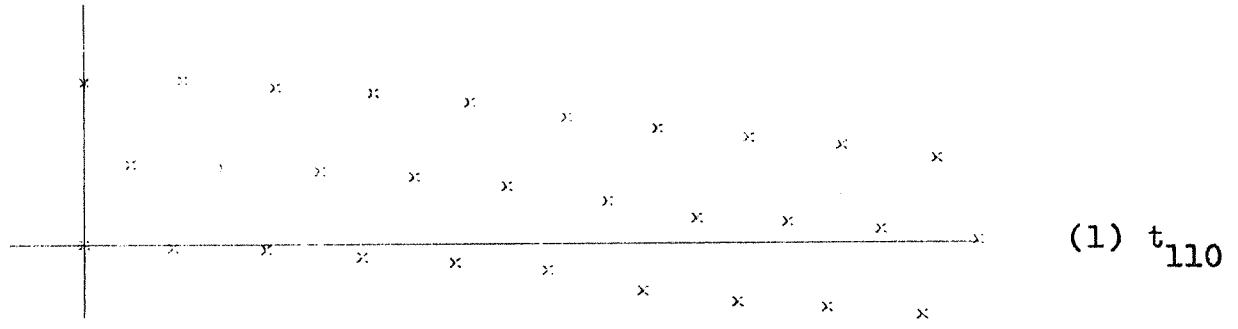
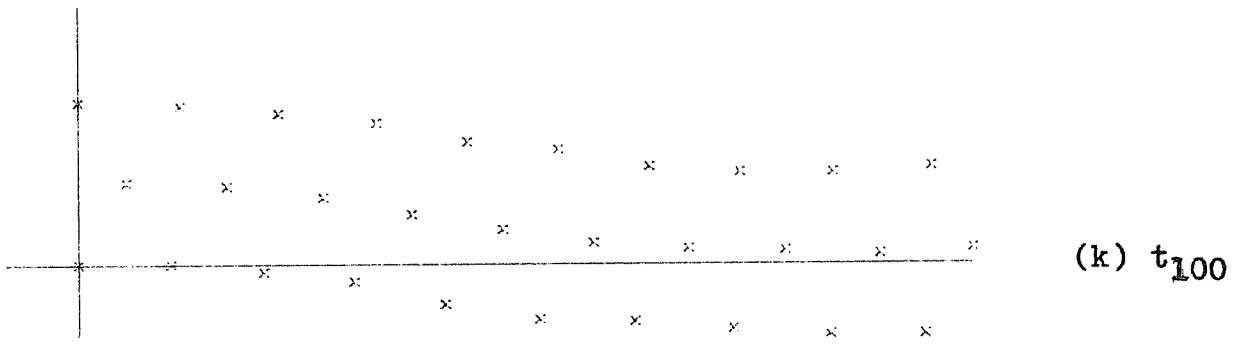
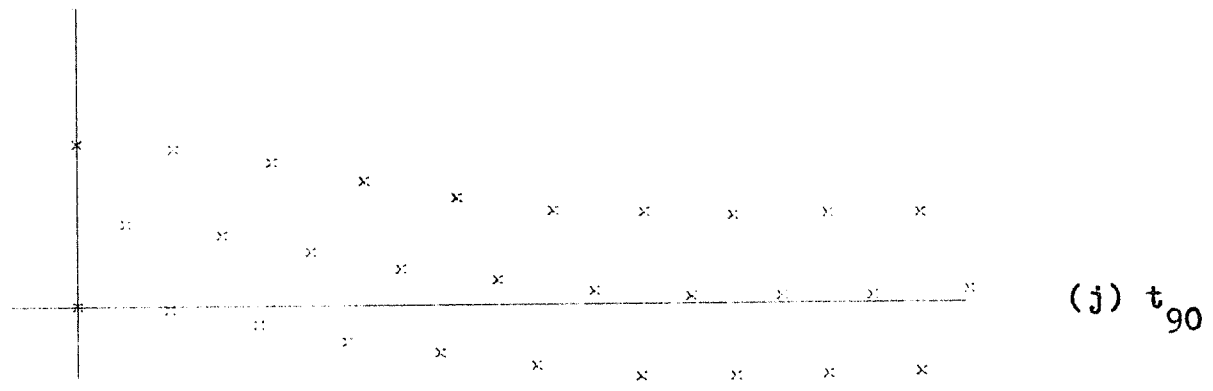
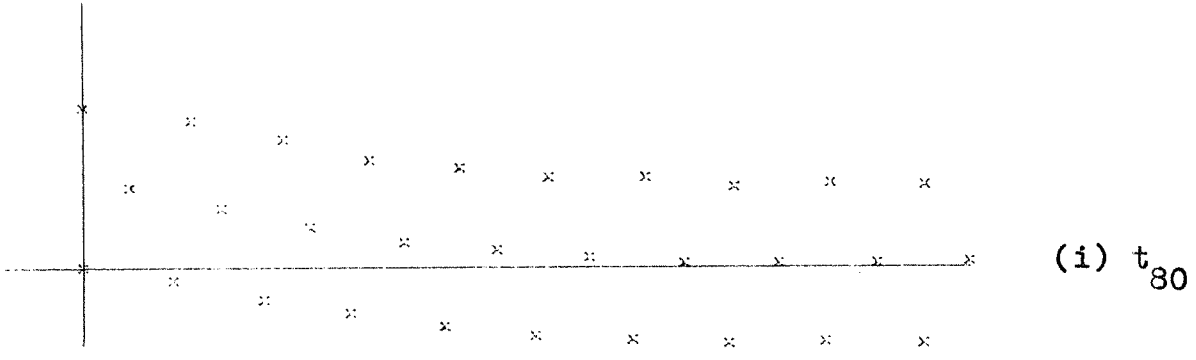


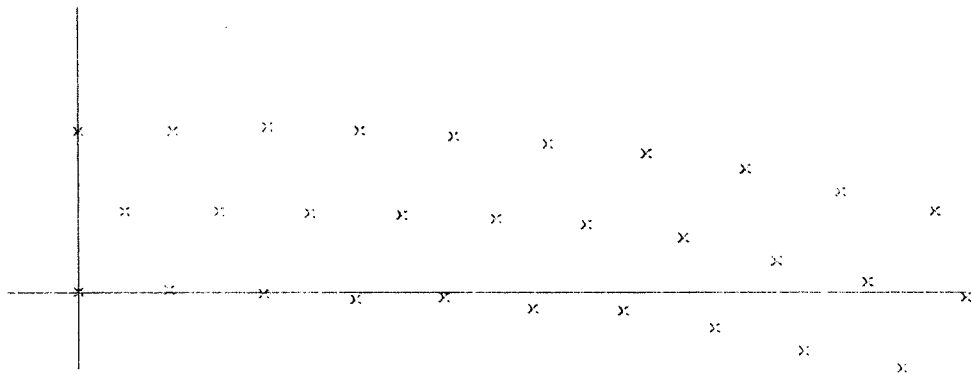
are to be held fixed throughout. In order to obtain an initial position of tension like that shown in Figure 4.1a, first set  $P_{13}, P_{14}, P_{15}, P_{16}, P_{17}, P_{18}, P_{19}, P_{20}$  and  $P_{21}$  at  $(2.28357, 1.29198), (3.80588, 1.26541), (5.32632, 1.18573), (6.84052, 1.02658), (8.33992, .76219), (9.81058, .36813), (11.23199, -.17750), (12.57631, -.89228),$  and  $(13.80807, -1.78721),$  respectively. Any two consecutive points  $P_k, P_{k+1}, k = 13, 14, \dots, 20,$  are positioned  $r$  units apart. The points  $P_2 - P_{10}$  and  $P_{23} - P_{31}$  are then positioned as follows:  $P_{k-10}$  and  $P_{k+11}$  are the two points which are  $r$  units from both  $P_k$  and  $P_{k+1}$  for each of  $k = 12, 13, \dots, 20.$  Each consecutive pair of points in the  $P_2 - P_{10}$  set is then separated by a distance greater than  $r,$  while each consecutive pair of points in the  $P_{23} - P_{31}$  set is separated by a distance less than  $r.$  Thus, the points  $P_2 - P_{10}$  are in a stretched position, while the points  $P_{23} - P_{31}$  are compressed.

From the initial position of tension shown in Figure 4.1a, the oscillatory motion of the bar is determined from (2.1)-(2.4) with all initial velocities set as  $\vec{0}.$  The upward swing of the bar was plotted automatically at every ten time steps and is shown in Figure 4.1a-w from  $t_0$  to  $t_{220}.$  It is of interest to note that as the bar moves, each row of particles exhibits wave oscillation and reflection.

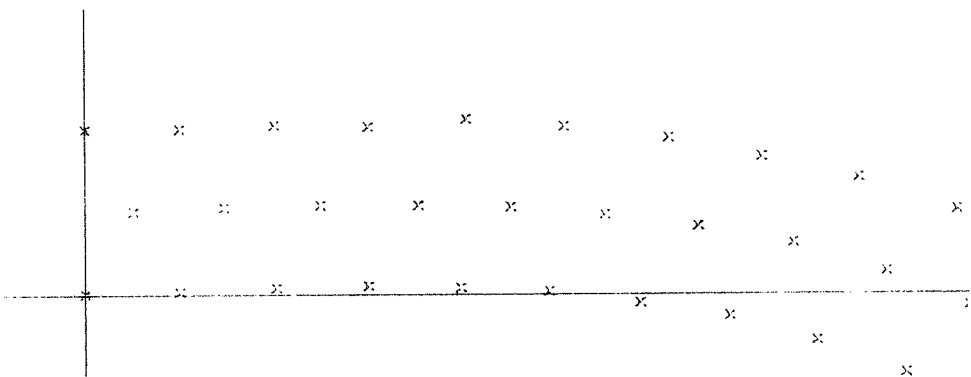




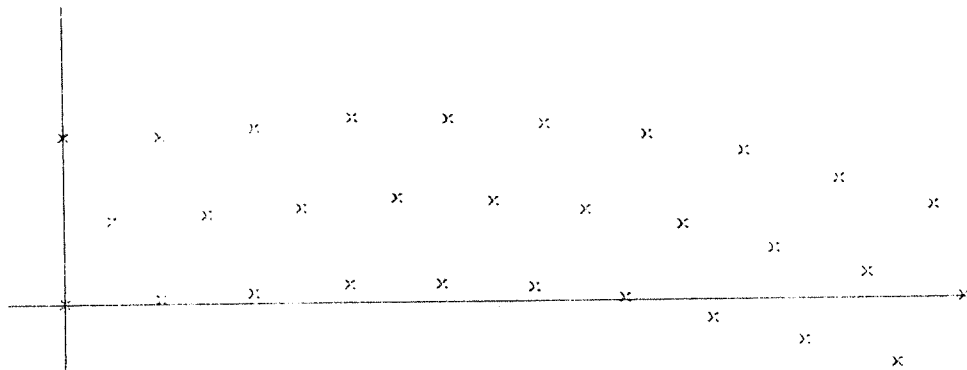




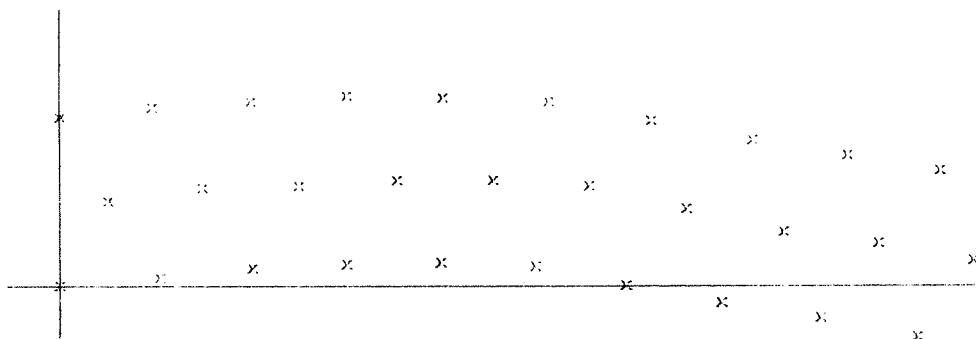
(m)  $t_{120}$



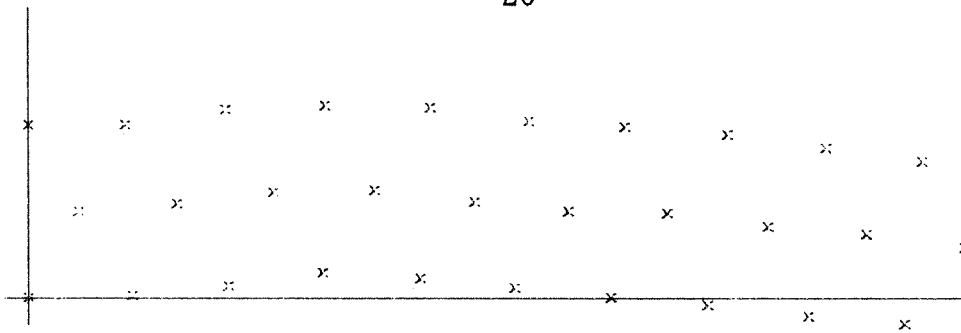
(n)  $t_{130}$



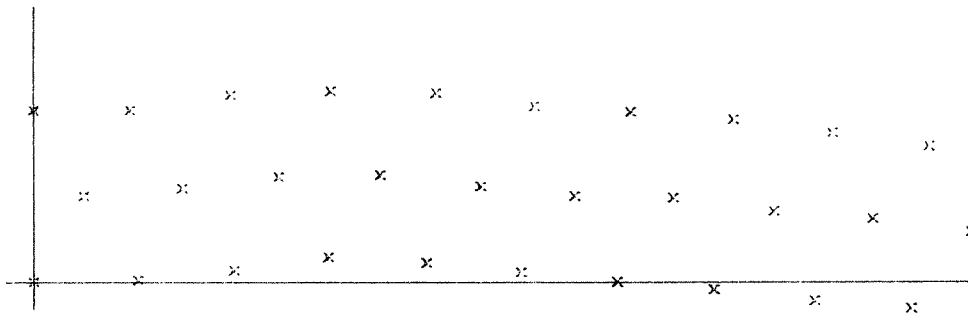
(o)  $t_{140}$



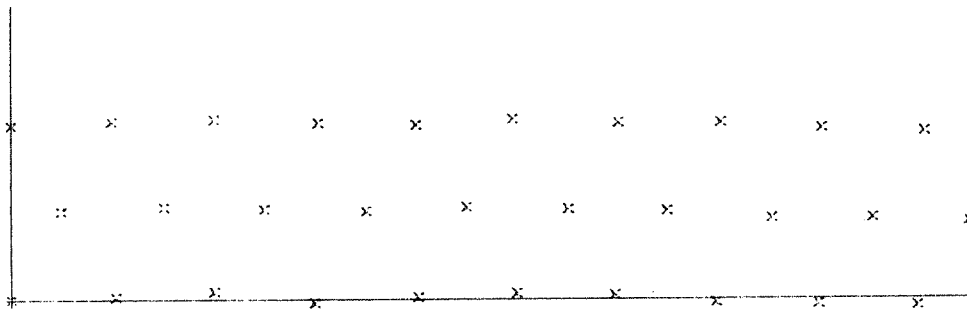
(p)  $t_{150}$



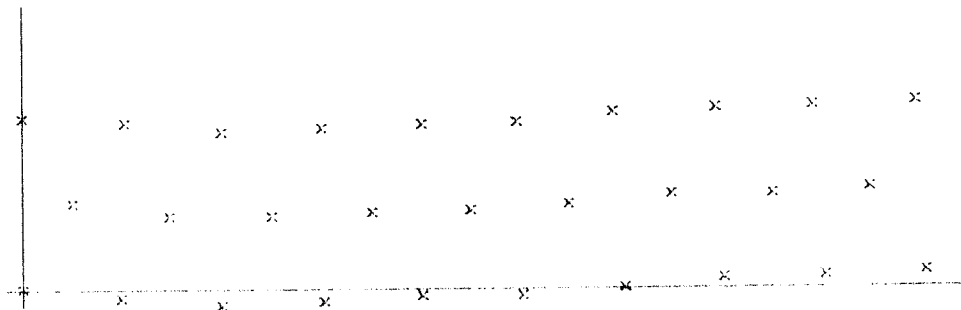
(q)  $t_{160}$



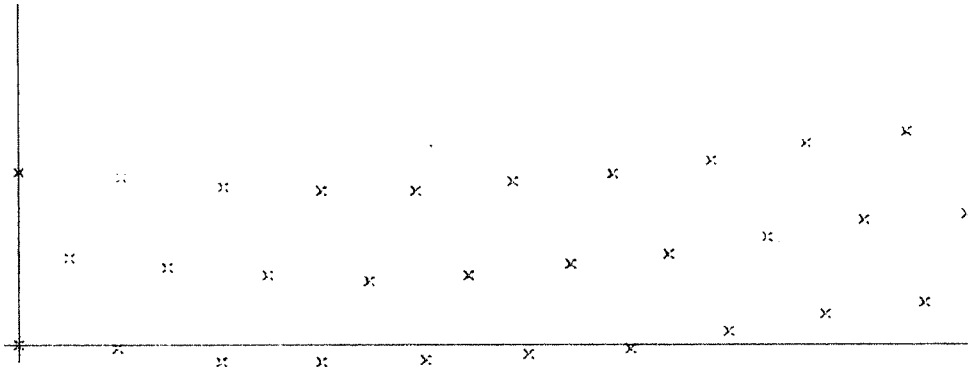
(r)  $t_{170}$



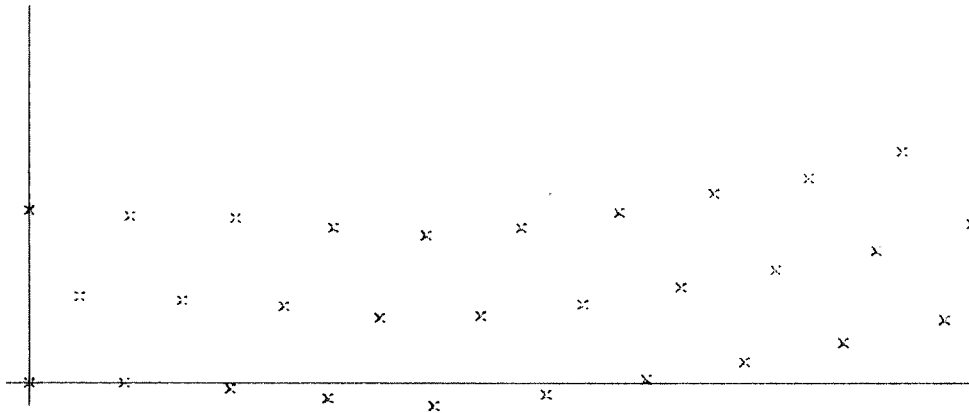
(s)  $t_{180}$



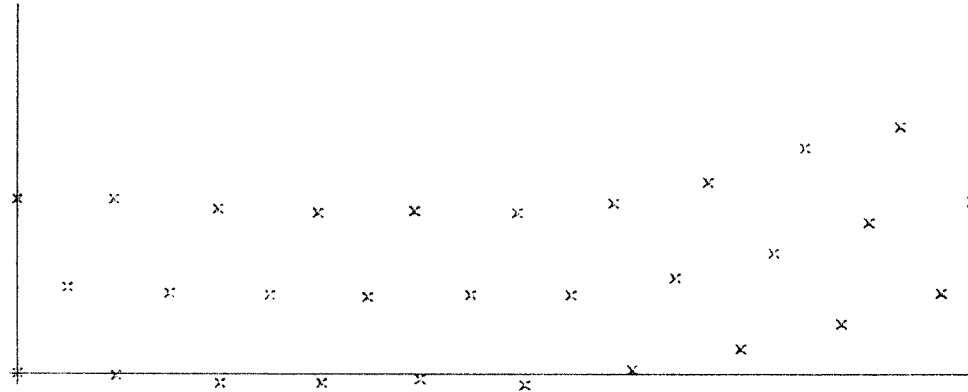
(t)  $t_{190}$



(u)  $t_{200}$



(v)  $t_{210}$



(w)  $t_{220}$

## 6. Remarks

A limited number of other examples were run, and these indicated that square building blocks were less stable than triangular ones, while the choices  $\alpha = 2, \beta = 5$  and  $\alpha = 7, \beta = 13$  were less viable than  $\alpha = 7, \beta = 10$ . Generally speaking, any choice  $p > q$  resulted in increased oscillations so that, for example, for the elastic bar model of Section 4, the choice  $p = 3, q = 1$  required a refinement of time step to  $\Delta t = 10^{-3}$  in order to study the resulting oscillations. The major handicap in all the computer examples run was the lack of adequate funding to enable the study of models with large numbers of particles.

Finally, it should be noted that the writer feels that varying  $\alpha, \beta, p$  and  $q$  in computer models with large numbers of particles will enable the researcher to produce viable computer models and to derive insight into the actual parameter values for various physical solids.



References

1. J. Douglas, Jr., "A Survey of Numerical Methods for Parabolic Differential Equations," in Advances in Computers, vol. 2, Academic Press, N. Y., 1961, pp. 1-55.
2. R. P. Feynman, R. B. Leighton, and M. Sands, The Feynman Lectures on Physics, Addison-Wesley, Reading, Mass., 1964.
3. A. Friedman, Partial Differential Equations of Parabolic Type, Prentice-Hall, Englewood Cliffs, N. J., 1964.
4. D. Greenspan, Lectures on the Numerical Solution of Linear, Singular, and Nonlinear Differential Equations, Prentice-Hall, Englewood Cliffs, N. J., 1968.
5. \_\_\_\_\_, "New Forms of Discrete Mechanics," Kybernetes, 1, 1972, pp. 87-101.
6. \_\_\_\_\_, "An Algebraic, Energy Conserving Formulation of Classical Molecular and Newtonian n-Body Interaction," to appear in Bull. Amer. Math. Soc.
7. S. T. Jones, "FORTRAN program for discrete conductive heat transfer," Appendix, TR#164, Department of Computer Sciences, Univ. of Wis., Madison, 1972.
8. C. Kittel, Introduction to Solid State Physics, 4th Ed., Wiley, N. Y., 1971.
9. P. G. Klemens, "Theory of Thermal Conductivity of Solids," in Thermal Conductivity I, Academic Press, N. Y., 1969, pp. 1-68.
10. S. S. Kutateladze and V. M. Borishanskii, A Concise Encyclopedia of Heat Transfer, Pergamon, N. Y., 1966.
11. V. V. Novozhilov, Foundations of the Nonlinear Theory of Elasticity, Graylock, Rochester, N. Y., 1953.
12. W. Prager, Introduction to the Mechanics of Continua, Ginn, Boston, 1961.

13. S. Timoshenko and J. Gere, Theory of Elastic Stability, McGraw-Hill, N. Y. , 1961.
14. S. Timoshenko and S. Woinowsky-Krieger, Theory of Plates and Shells, McGraw-Hill, N. Y. , 1959.

APPENDIX: FORTRAN Program for Discrete Conductive Heat Transfer  
by S. T. Jones

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C  FORTRAN PROGRAM FOR DISCRETE CONDUCTIVE HEAT TRANSFER
C
C  INDEX TO PROGRAM VARIABLES
C
C  A = ALPHA
C  ANGLE(I) = ANGLE (IN DEGREES) OF INITIAL VELOCITY VECTOR WITH
C             RESPECT TO POSITIVE X-AXIS
C  B = BETA
C  DD(I) = DISTANCE OF PARTICLE I FROM ITS INITIAL POSITION
C  DMAX = MAXIMUM ALLOWABLE DISTANCE OF ANY PARTICLE FROM ITS INITIAL
C         POSITION
C  DT = TIME INCREMENT
C  EPS = CONVERGENCE CRITERION FOR NEWTON'S METHOD
C  FX(I) = FORCE COMPONENT ON PARTICLE I IN X-DIRECTION
C  FY(I) = FORCE COMPONENT ON PARTICLE I IN Y-DIRECTION
C  IAXIS(I) = 0 IF LEFTMOST PARTICLE IN ROW I IS ON Y-AXIS
C             = +1,-1 IF LEFTMOST PARTICLE TO BE SHIFTED RIGHT OR LEFT,
C             RESPECTIVELY
C  IEND = 0 IF ANOTHER DATA CASE FOLLOWS
C         = 1 IF END OF RUN
C  IMAX = MAXIMUM NUMBER OF ITERATIONS PER TIMESTEP FOR NEWTON'S
C         METHOD
C  IPRINT = PRINT-STEP INCREMENT
C  IPUNCH = PUNCH-STEP INCREMENT
C  IROW(I) = NUMBER OF PARTICLES IN ROW I
C  ISTART = 0 IF NEW DATA CASE
C           = 1 IF RESTART
C  IVFL(I) = NUMBER OF PARTICLE TO BE GIVEN AN INITIAL VELOCITY
C  JPUNCH = 0 IF NO PUNCH REQUIRED, PUNCH OTHERWISE
C  M(I) = MASS OF PARTICLE I
C  N = NUMBER OF PARTICLES IN SYSTEM
C  NAXIS = NUMBER OF ROW ON X-AXIS
C  NFIX = TOTAL NUMBER OF PARTICLES TO BE FIXED
C  NMAX = MAXIMUM NUMBER OF TIMESTEPS THIS DATA CASE
C  NP(I) = NUMBER OF PARTICLE TO BE FIXED
C  NROW = NUMBER OF ROWS IN SYSTEM
C  NSTP = TIMESTEP NUMBER
C  NVFL = TOTAL NUMBER OF PARTICLES TO BE GIVEN AN INITIAL VELOCITY
C  OMEGA = SUCCESSIVE OVER-RELAXATION FACTOR FOR NEWTON'S METHOD
C  P = ATTRACTION PARAMETER
C  Q = REPULSION PARAMETER
C  R(I,J,1) = DISTANCE BETWEEN PARTICLES I AND J AT PREVIOUS TIMESTEP
C  R(I,J,2) = DISTANCE BETWEEN PARTICLES I AND J AT CURRENT TIMESTEP
C  SKF(I) = SUM OF KINETIC ENERGIES FOR PARTICLE I OVER ALL TIMESTEPS
C  TEMP(I) = MEASURE OF TEMPERATURE OF PARTICLE I AT CURRENT TIMESTEP
C  VEL(I) = MAGNITUDE OF INITIAL VELOCITY VECTOR
C  VX(I,1) = X-COMPONENT OF VELOCITY OF PARTICLE I, PREVIOUS TIMESTEP
C  VX(I,2) = SAME AS ABOVE, CURRENT TIMESTEP, PREVIOUS ITERATION
C  VX(I,3) = SAME AS ABOVE, CURRENT TIMESTEP, CURRENT ITERATION
C  VX0(I) = X-COMPONENT OF INITIAL VELOCITY, PARTICLE I
C  VY(J,1) = Y-COMPONENT OF VELOCITY OF PARTICLE I,
C  VY(I,2) = SAME DEFINITIONS
C  VY(I,3) = AS VX(I,J), ABOVE
C  VY0(I) = Y-COMPONENT OF INITIAL VELOCITY, PARTICLE I

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C      X(I,1) = X-COMPONENT OF POSITION OF PARTICLE I,
C      X(I,2) =      SAME DEFINITIONS
C      X(I,3) =      AS VX(I,J)
C      XU(I) = X-COMPONENT OF INITIAL POSITION, PARTICLE I
C      XKE(I) = KINETIC ENERGY OF PARTICLE I AT EACH TIMESTEP
C      Y(I,1) = Y-COMPONENT OF POSITION OF PARTICLE I,
C      Y(I,2) =      SAME DEFINITIONS
C      Y(I,3) =      AS VX(I,J)
C      YU(I) = Y-COMPONENT OF INITIAL POSITION, PARTICLE I
C
      IMPLICIT DOUBLE PRECISION(A-H,M,O-Z)
      DIMENSION XU(100),YU(100),VXU(100),VYU(100),X(100,3),Y(100,3),
      VX(100,3),VY(100,3),FX(100),FY(100),DD(100),XKE(100),NP(100),
      ZR(45,45,2),M(100)
      DIMENSION IROW(5),IAXIS(5),IVEL(5),VEL(5),ANGLE(5)
      DIMENSION SKF(50),TEMP(50)
1001  FORMAT(2010.0)
1002  FORMAT(16I5)
1003  FORMAT(4010.0,I5)
1004  FORMAT(I5,2010.0)
2000  FORMAT(1H1)
2001  FORMAT(5X,'N',5X,'OMEGA',5X,'EPS',5X,'IMAX',/,I7,F10.4,E8.1,I9)
2002  FORMAT(/,6X,'A',8X,'B',8X,'P',8X,'Q',7X,'DT',/,4F9.3,E9.2)
2003  FORMAT(' TIMESTEP ',6X,'M',9X,'X',14X,'Y',13X,'DD',13X,'XKE',13X,
      1'SKE',13X,'TEMP',/,2X,I6)
2004  FORMAT(10X,F6.4,4F15.10)
2005  FORMAT(' NON-CONVERGENCE AFTER ',I3,' ITERATIONS FOR TIMESTEP= ',
      1I6,' DT= ',F9.2)
2006  FORMAT(' OSCILLATION BEYOND MAXIMUM ALLOWABLE LIMIT OF ',F6.4)
2501  FORMAT(2025,18)
      PRINT 2000
      NSTP=0
      READ 1001,OMEGA,EPS,DMAX,DT
      READ 1002,NMAX,IMAX,IPRINT,IPUNCH,JPUNCH,ISTART
      READ 1003,N,NROW,NFIX,NAXIS,NVEL
      READ 1004,A,B,P,Q,TEND
      PRINT 2001,N,OMEGA,EPS,IMAX
      DO 20 I=1,N
      XKE(I)=0.0
      DD(I)=0.0
      SKF(I)=0.0
      TEMP(I)=0.0
      20  CONTINUE
      IF(ISTART.EQ.0)GO TO 1
C  RESTART
      READ 1002,NSTP
      READ 2501,(XU(I),YU(I),I=1,N)
      READ 2501,(VXU(I),VYU(I),I=1,N)
      READ 2501,(SKF(I),I=1,N)
      GO TO 5
C  NEW CASE--CALCULATE POSITIONS
      1  BASEFX=(Q*(B-1)/(P*(A-1)))*(1.0/(B-A))
      BASEY=SQRT(BASEFX**2-(0.5*BASEFX)**2)
      READ 1002,(IROW(I),IAXIS(I),I=1,NROW)

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      TL=1
      IU=0
      DO 3 I=1,NROW
      IU=IU+TR0W(I)
      XSHIFT=I*.5*TA*IS(T)*BASEX
      USHIFT=NAXIS-I
      DO 2 J=IL,IU
      XD(J)=(J-TL)*BASEX+XSHIFT
      YD(J)=USHIFT*BASEY
  2 CONTINUE
      IL=IU+1
  3 CONTINUE
C NEW CASE--CALCULATE VELOCITIES
10 DO 4 I=1,N
      VX0(I)=0.0
      VY0(I)=0.0
  4 CONTINUE
      IF(NVEL.EQ.0) GO TO 6
      READ 1004,(IVFL(I),VFL(I),ANGLE(I),I=1,NVFL)
      PI=3.141592653589793240*00
      RAD=PI/180.0
      DO 5 I=1,NVFL
      J=IVFL(I)
      THETA=ANGLE(I)*RAD
      VX0(J)=VFL(I)*COS(THETA)
      VY0(J)=VFL(I)*SIN(THETA)
  5 CONTINUE
  6 DO 7 I=1,N
      M(I)=1.0
  7 CONTINUE
      OMW=1.0-OMEGA
      IF(NFIX.EQ.0) GO TO 11
      READ 1002,(NF(I),I=1,NFIX)
11 PRINT 2002,A,B,P,Q,DT
      T=0.0
      DT2=DT/2.0
      PRINT 2003,NSTP
      DO 30 I=1,N
      PRINT 2004,M(I),XD(I),YD(I),DD(I),XKF(I)
30 CONTINUE
C SPECIFY INITIAL GUESS FOR NEWTON'S ITERATION AT FIRST TIMESTEP
DO 40 I=1,N
      X(I,3)=XD(I)
      VX(I,3)=VX0(I)
      Y(I,3)=YD(I)
40 VY(I,3)=VY0(I)
      CALL RCALC
C UPDATE POSITIONS,VELOCITIES,DISTANCES FOR ALL TIMESTEPS
45 NSTP=NSTP+1
      T=T+DT
      DO 60 J=1,N
      X(I,1)=X(I,3)
      VX(I,1)=VX(I,3)
      Y(I,1)=Y(I,3)

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      VY(I,1)=VY(I,3)
      DO 50 J=1,N
50 R(T,J,1)=R(I,J,2)
60 CONTINUE
C BEGIN ITERATION LOOP
      DO 90 K=1,IMAX
C UPDATE ALL VARIABLES, CURRENT TIMESTEP, PREVIOUS ITERATION
      DO 70 I=1,N
        X(I,2)=X(I,3)
        VX(I,2)=VX(I,3)
        Y(I,2)=Y(I,3)
        VY(I,2)=VY(I,3)
      70 CONTINUE
C UPDATE POSITIONS, CURRENT TIMESTEP, CURRENT ITERATION
      DO 73 I=1,N
        IF(NFIX.EQ.0)GO TO 72
        DO 71 J=1,NFIX
          IF(I.EQ.NP(J))GO TO 73
        71 CONTINUE
        72 X(I,3)=OMW*X(I,2)+OMEGA*(DT2*(VX(I,2)+VX(I,1))+X(I,1))
          Y(I,3)=OMW*Y(I,2)+OMEGA*(DT2*(VY(I,2)+VY(I,1))+Y(I,1))
        73 CONTINUE
        CALL RCALC
        CALL FCALC
C UPDATE VELOCITIES, CURRENT TIMESTEP, CURRENT ITERATION
      DO 80 I=1,N
        IF(NFIX.EQ.0)GO TO 75
        DO 74 J=1,NFIX
          IF(I.EQ.NP(J))GO TO 80
        74 CONTINUE
        75 VX(I,3)=OMW*VX(I,2)+OMEGA*(DT*FX(I)+VX(I,1))
          VY(I,3)=OMW*VY(I,2)+OMEGA*(DT*FY(I)+VY(I,1))
        80 CONTINUE
C TEST FOR CONVERGENCE
      DO 85 I=1,N
        IF(ABS(X(I,3)-X(I,2)).GT.FPS)GO TO 90
        IF(ABS(Y(I,3)-Y(I,2)).GT.FPS)GO TO 90
        IF(ABS(VX(I,3)-VX(I,2)).GT.FPS)GO TO 90
        IF(ABS(VY(I,3)-VY(I,2)).GT.FPS)GO TO 90
      85 CONTINUE
      GO TO 95
      90 CONTINUE
      PRINT 2005,K,NSTP
      GO TO 110
      95 CALL DCALC
      DO 100 I=1,N
        IF(DD(I).GT.DMAX)GO TO 105
100 CONTINUE
      STFP=NSTP
      DO 700 II=1,N
        XKF(II)=0.5*M(II)*(VX(II,3)*VX(II,3)+VY(II,3)*VY(II,3))
        SKF(II)=SKF(II)+XKF(II)
        IFMP(II)=SKF(II)/STFP
700 CONTINUE

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      IF (MOD(NSTP, IPRINT), NE, 0) GO TO 103
      IF (JPUNCH, EQ, 0) GO TO 102
      IF (MOD(NSTP, IPUNCH), NE, 0) GO TO 102
      WRITE(1, 1002) NSTP
      DO 151 I=1, N
      WRITE(1, 2501) X(I, 3), Y(I, 3)
151  CONTINUE
      DO 152 I=1, N
      WRITE(1, 2501) VX(I, 3), VY(I, 3)
152  CONTINUE
      WRITE(1, 2501) (SKF(II), II=1, N)
102  CALL OUTP
103  IF (NSTP, EQ, NMAX) GO TO 110
      GO TO 45
105  PRINT 2005, OMAX
      CALL OUTP
110  IF (IEND, EQ, 0) GO TO 10
      STOP
C   INTERNAL SUBROUTINE TO COMPUTE DISTANCES BETWEEN PARTICLES
      SUBROUTINE R0CALC
      DO 210 II=1, N
      IP1=II+1
      DO 200 JJ=IP1, N
      R(II, JJ, 2)=SQRT((X(II, 3)-X(JJ, 3))**2+(Y(II, 3)-Y(JJ, 3))**2)
      R(JJ, II, 2)=R(II, JJ, 2)
200  CONTINUE
210  CONTINUE
      RETURN
C   INTERNAL SUBROUTINE TO COMPUTE DISTANCE OF PARTICLE FROM ITS INITIAL
C   POSITION
      SUBROUTINE D0CALC
      DO 300 II=1, N
      DD(II)=SQRT((X(II, 3)-X0(II))**2+(Y(II, 3)-Y0(II))**2)
300  CONTINUE
      RETURN
C   INTERNAL SUBROUTINE TO COMPUTE FORCES
      SUBROUTINE F0CALC
      IA=A-1
      IR=B-1
      DO 600 II=1, N
      IF (NFTX, EQ, 0) GO TO 450
      DO 400 KK=1, NFTX
      IF (II, EQ, NP(KK)) GO TO 600
400  CONTINUE
450  SUMX=0.0
      SUMY=0.0
460  DO 500 JJ=1, N
      IF (II, EQ, JJ) GO TO 550
      SUMP=0.0
      SUMQ=0.0
      RTJ=R(II, JJ, 1)+R(II, JJ, 2)
      DO 500 IZ=1, IA
      SUMP=SUMP+(R(II, JJ, 1)**(IZ-1))*R(II, JJ, 2)**(A-(IZ-1)-2))
500  CONTINUE

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      DO 501 IZ=1,IR
      SUMQ=SUMQ+(R(II,JJ,1)**(IZ-1))*R(II,JJ,2)**(R-(IZ-1)-2))
501  CONTINUE
      PD=R(II,JJ,1)**(A-1)*R(II,JJ,2)**(A-1)*RIJ
      SUMP=F*SUMP/PD
      QD=R(II,JJ,1)**(B-1)*R(II,JJ,2)**(B-1)*RIJ
      SUMQ=Q*SUMQ/QD
      SUMX=(SUMQ-SUMP)*M(JJ)*(X(II,3)+X(II,1)-X(JJ,3)-X(JJ,1))+SUMX
      SUMY=(SUMQ-SUMP)*M(JJ)*(Y(II,3)+Y(II,1)-Y(JJ,3)-Y(JJ,1))+SUMY
550  CONTINUE
      FX(II)=SUMX
      FY(II)=SUMY
600  CONTINUE
      RETURN
C  INTERNAL PRINT SUBROUTINE
      SUBROUTINE GUTP
3001  FORMAT(2X,T6)
3002  FORMAT(1X,6F15.10)
      PRINT 3001,NSTP
      DO 800 II=1,N
      PRINT 3002,X(II,3),Y(II,3),QD(II),XKF(II),SKF(II),TEMP(II)
800  CONTINUE
      RETURN
      END

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